

## CM3120: Module 4

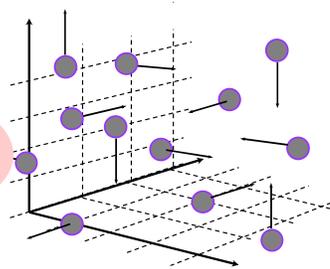
### Diffusion and Mass Transfer II

- I. Mass transfer in distillation and absorption
  - A. Film model
  - B. Penetration model
- II. Linear driving force model (mass transfer coefficient,  $k_x$ )
  - A. Review: no bulk convection
  - B. New: appreciable bulk convection
  - C. Predict mass transfer coefficients
  - D. Solve unsteady mass transfer problems
- III. Macroscopic species A mass balances
- IV. Dimensional analysis in mass transfer
  - A. Review—compare to heat
  - B. Engineering quantities of interest
  - C. Data correlations for  $k_x$  (Sh or  $Nu_{AB}$  correlations)
- V. Overall mass transfer coefficients

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## CM3120: Module 4

Module 4 Lecture IV  
**Dimensional Analysis  
 in Mass Transfer**



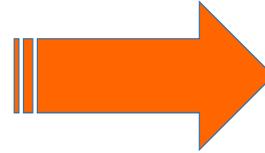
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[www.chem.mtu.edu/~fmorriso/cm3120/cm3120.html](http://www.chem.mtu.edu/~fmorriso/cm3120/cm3120.html)

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Continuing work with the linear driving force for mass transfer, i.e. mass transfer coefficients,  $k_c$



Linear Driving Force Model for Mass Transfer

CM3110  
Transport II  
Part II: Diffusion and Mass Transfer

Michigan Tech

**Linear Driving Force Model for Mass Transfer**

$$|N_A| = k_y |y_{A,bulk} - y_{A,i}|$$

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Mass Transport "Laws"

We have 2 Mass Transport "laws"

Remaining Topics to round out our understanding of mass transport:

Fick's law of diffusion

- $D_{AB}$
1. Since we predict  $N_A$  with Fick's law, we can also predict a mass transfer coefficients  $k_y$  or  $k_c$  *Relate  $k_c$  and  $D_{AB}$*
  2. 1D Unsteady models can be solved (if good at math) *Solutions are analogous to heat transfer*

Mass transfer coefficients

- $k_c$
3. Combine with macroscopic species A mass balance *Model macroscopic processes, design units*
  4. Are not material properties; rather, they are determined experimentally and specific to the situation (dimensional analysis and correlations)
  5. Facilitate combining resistances into overall mass transfer coefficients,  $K_L, K_G$ , to be used in modeling unit operations

Mass Transport "Laws"

We now have 2 Mass Transport "laws"

**Fick's Law of Diffusion**  $N_A = x_A(N_A + N_B) - cD_{AB} \nabla x_A$  Transport coefficient

Use: Combine with microscopic species A mass balance  
Predicts flux  $N_A$  and composition distributions, e.g.  $x_A(x, y, z, t)$   
1D Steady models can be solved  
1D Unsteady models can be solved (if good at math) ②  
2D steady and unsteady models can be solved by COMSOL  
Since we predict  $N_A$ , we can also predict a mass xfer coeff  $k_y$  or  $k_c$  ①  
Diffusion coefficients are **material** properties (see tables)

**Linear-Driving-Force Model**  $|N_A| = k_y |y_{A,bulk} - y_{A,i}|$

Use: Combine with macroscopic species A mass balance ③  
Predicts flux  $N_A$ , but **not** composition distributions  
May be used as a boundary condition in microscopic balances  
Mass-transfer-coefficients are **not material properties**  
Rather, they are determined experimentally and specific to the situation (dimensional analysis and correlations) ④  
Facilitate combining resistances into overall mass xfer coeffs,  $K_L, K_G$  ⑤



Dimensional Analysis in Mass Transfer



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CM3110  
Transport II  
Part II: Diffusion and Mass Transfer

**Dimensional Analysis  
in Mass Transfer**

**“D.A.”**

④



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Dimensionless Numbers	
Re – Reynolds = $\frac{\rho v D}{\mu}$ Fr – Froude = $\frac{v^2}{g D}$ Pe – Péclet <sub>h</sub> = RePr = $\frac{\rho v D^2}{\alpha}$ Pe – Péclet <sub>m</sub> = ReSc = $\frac{\rho v D^2}{D_{AB}}$	These numbers from the governing equations tell us about the relative importance of the terms they precede in the microscopic balances ( <b>scenario properties</b> ).
Pr – Prandtl = $\frac{c_p \mu}{k}$ Sc – Schmidt = $\frac{\mu}{\rho D_{AB}}$ Le – Lewis = $\frac{\alpha}{D_{AB}}$	These numbers compare the magnitudes of the diffusive transport coefficients $\nu, \alpha, D_{AB}$ ( <b>material properties</b> ).
f – Friction Factor = $\frac{F_{wall}}{(\frac{\rho v^2}{2})_{Ac}}$ Nu – Nusselt = $\frac{h D}{k}$ Sh – Sherwood = $\frac{h_m D}{D_{AB}}$	These numbers are defined to help us build transport data correlations based on the fewest number of grouped (dimensionless) variables ( <b>scenario properties</b> ).
St <sub>k</sub> = Nu/Pe <sub>h</sub> , St <sub>m</sub> = Sh/Pe <sub>m</sub> – Stanton	

mass transfer?

heat transfer?

**What do we do to understand complex flows?**

**Same strategy as:**

**flows**

- Turbulent tube flow
- Noncircular conduits
- Drag on obstacles
- Boundary Layers

**heat transfer**

- Forced-convection heat transfer coefficients
- Natural-convection heat transfer coefficients
- Problems with multiple kinds of physics

1. Find a simple problem that allows us to identify the physics
2. Nondimensionalize
3. Explore that problem
4. Take data and correlate
5. Solve real problems

**Solve Real Problems.**

**Powerful.**

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Solve Real Problems. Powerful.

**mass transfer?**  
~~heat transfer?~~  
~~flows?~~

**What do we do to understand complex flows?**

**Same strategy as:**

**flows**

- Turbulent tube flow
- Noncircular conduits
- Drag on obstacles
- Boundary Layers

**heat transfer**

- Forced-convection heat transfer coefficients
- Natural-convection heat transfer coefficients
- Problems with multiple kinds of physics

**Mass transfer**

- From fluid to plate
- To a falling film
- In pipes and ducts
- Past submerged objects
- To/from bubbles, drops
- In agitated systems
- In fixed and fluidized beds
- In packed 2-phase contactors (absorption, distillation, cooling towers)

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Dimensional Analysis in Mass Transfer

**Let's review our review of dimensional analysis...**

**heat transfer**

Heat Transfer: Steady vs. Unsteady

**CM3120, Lecture 5**

**What is our usual strategy for complex phenomena?**

**Answer: Dimensional Analysis**

**CM3110: Momentum and Heat Xfer**

Complex Heat Transfer – Dimensional Analysis

**Experience with Dimensional Analysis (momentum):**

- Flow in pipes at all flow rates (laminar and turbulent)  
Solution: Navier-Stokes, Re, Fr, L/D, dimensionless wall force =  $f$ ;  $f = f(Re, L/D)$
- Rough pipes  
Solution: add additional length scale; then nondimensionalize
- Non-circular conduits  
Solution: Use hydraulic diameter as the length scale of the flow to nondimensionalize
- Flow around obstacles (spheres, other complex shapes)  
Solution: Navier-Stokes, Re, dimensionless drag =  $C_D$ ;  $C_D = C_D(Re)$
- Boundary layers  
Solution: Two components of velocity need independent length scales

**Let's review**

➔

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Dimensional Analysis in Mass Transfer Review of dimensional analysis

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Complex Heat Transfer (CM3110) CM3110 REVIEW

**How do we handle complex geometries, complex flows, complex machinery?**

Process scale

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Dimensional Analysis in Mass Transfer Review of dimensional analysis

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Complex Heat Transfer – Dimensional Analysis CM3110 REVIEW

*(Answer: Use the same techniques we have been using in fluid mechanics)*

**Engineering Modeling (complex systems)**

- Choose an idealized problem and solve it
- From insight obtained from **ideal** problem, identify governing equations of **real** problem
- Nondimensionalize the governing equations; deduce dimensionless scale factors (e.g. Re, Fr for fluids)
- Design experiments to test modeling thus far
- Revise modeling (structure of dimensional analysis, identity of scale factors, e.g. add roughness lengthscale)
- Design additional experiments
- Iterate until useful correlations result

Process scale

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Dimensional Analysis in Mass Transfer
Review of dimensional analysis

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momentum transfer

Complex Heat Transfer – Dimensional Analysis

CM3110  
REVIEW

Experience with Dimensional Analysis (momentum):

- Flow in pipes at all flow rates (laminar and turbulent)  
**Solution:** Navier-Stokes,  $Re$ ,  $Fr$ ,  $L/D$ ,  
 dimensionless drag =  $f$ ;  $f = f(Re, L/D)$
- Rough pipes  
**Solution:** add additional length scale; then  
 nondimensionalize
- Non-circular conduits  
**Solution:** Use hydraulic diameter as the length  
 scale of the flow to nondimensionalize
- Flow around obstacles (spheres, other complex shapes)  
**Solution:** Navier-Stokes,  $Re$ , dimensionless  
 drag =  $C_D$ ;  $C_D = C_D(Re)$
- Boundary layers  
**Solution:** Two components of velocity  
 need independent lengthscales

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Dimensional Analysis in Mass Transfer
Review of dimensional analysis

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### Correlations compared with data

momentum transfer

Turbulent flow (smooth pipe)

Rough pipe

Noncircular cross section

Around obstacles

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Correlations compared with data

These have been impressive victories for dimensional analysis

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Heat Transfer: Steady vs. Unsteady

CM3110 REVIEW

How did Dimensional Analysis work for steady heat transfer?

Answer: Here's the method:

- Choose an idealized problem and solve it
- From insight obtained from **ideal** problem, identify governing equations of **real** problem
- Nondimensionalize the governing equations; deduce dimensionless scale factors (e.g. Re, Fr for fluids)
- Design experiments to test modeling thus far
- Revise modeling (structure of dimensional analysis, identity of scale factors, e.g. add roughness lengthscale)
- Design additional experiments
- Iterate until useful correlations result

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Dimensional Analysis in Mass Transfer Review of dimensional analysis

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Choose "typical" or "characteristic" values; can only know if they are the right choices if the D.A. works.

steady heat transfer

**Forced Convection Heat Transfer**

CM3110 REVIEW

**Pipe flow**

z-component of the Navier-Stokes Equation:

$$\rho \left( \frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial P}{\partial z} + \mu \left( \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right) + \rho g_z$$

**Choose:**

**D** = characteristic length  
**V** = characteristic velocity  
**D/V** = characteristic time  
 **$\rho V^2$**  = characteristic pressure

- Choose "typical" values (*scale factors*)
- Use them to scale the equations
- Deduce which terms dominate

Choose "characteristic" values

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Choose "typical" or "characteristic" values; can only know if they are the right choices if the D.A. works.

steady heat transfer

**Forced Convection Heat Transfer**

CM3110 REVIEW

**Pipe flow**

non-dimensional variables:

time: $t^* \equiv \frac{tV}{D}$	position: $r^* \equiv \frac{r}{D}$  $z^* \equiv \frac{z}{D}$	velocity: $v_z^* \equiv \frac{v_z}{V}$  $v_r^* \equiv \frac{v_r}{V}$  $v_\theta^* \equiv \frac{v_\theta}{V}$	driving force: $P^* \equiv \frac{P}{\rho V^2}$  $g_z^* \equiv \frac{g_z}{g}$
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- Choose "typical" values (*scale factors*)
- Use them to scale the equations
- Deduce which terms dominate

Choose "characteristic" values

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Oops, re-used the "\*" notation; here it is dimensionless variable, not molar average velocity

steady heat transfer

**Forced Convection Heat Transfer**

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**Pipe flow** non-dimensional variables:

time: $t^* \equiv \frac{tV}{D}$	position: $r^* \equiv \frac{r}{D}$ $z^* \equiv \frac{z}{D}$	velocity: $v_z^* \equiv \frac{v_z}{V}$ $v_r^* \equiv \frac{v_r}{V}$ $v_\theta^* \equiv \frac{v_\theta}{V}$	driving force: $P^* \equiv \frac{P}{\rho V^2}$ $g_z^* \equiv \frac{g_z}{g}$
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- Choose "typical" values (scale factors)
- Use them to scale the equations
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Choose "characteristic" values

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Choose "typical" or "characteristic" values; can only know if they are the right choices if the D.A. works.

steady heat transfer

**Forced Convection Heat Transfer**

CM3110 REVIEW

**Energy**

Microscopic energy balance:

$$\rho \hat{c}_p \left( \frac{\partial T}{\partial t} + v_r \frac{\partial T}{\partial r} + \frac{v_\theta}{r} \frac{\partial T}{\partial \theta} + v_z \frac{\partial T}{\partial z} \right) = k \left( \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 T}{\partial \theta^2} + \frac{\partial^2 T}{\partial z^2} \right) + S$$

non-dimensional variables:

position: $r^* \equiv \frac{r}{D}$ $z^* \equiv \frac{z}{D}$	temperature: $T^* \equiv \frac{T - T_0}{(T_1 - T_0)}$	source: $S^* \equiv \frac{S}{S_0}$
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Choose:  
 T – use a characteristic interval (since distance from T = 0K is not part of this physics)  
 S – use a reference source, S<sub>0</sub>

$S_0 \equiv \frac{(T_1 - T_0) V \rho \hat{c}_p}{D} [=] \frac{W}{m^2}$

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Review of dimensional analysis

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**Micro E-Balance produces  $Pe = PrRe$**

steady heat transfer

Complex Heat Transfer – Dimensional Analysis

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REVIEW

Non-dimensional Energy Equation

$$\left(\frac{\partial T^*}{\partial t^*} + v_r^* \frac{\partial T^*}{\partial r^*} + \frac{v_\theta^*}{r^*} \frac{\partial T^*}{\partial \theta} + v_z^* \frac{\partial T^*}{\partial z^*}\right) = \frac{1}{Pe} \left( \frac{1}{r^*} \frac{\partial}{\partial r^*} \left( r^* \frac{\partial T^*}{\partial r^*} \right) + \frac{1}{r^{*2}} \frac{\partial^2 T^*}{\partial \theta^2} + \frac{\partial^2 T^*}{\partial z^{*2}} \right) + S^*$$

Non-dimensional Navier-Stokes Equation

$$\frac{Dv_z^*}{Dt^*} = -\frac{\partial P^*}{\partial z^*} + \frac{1}{Re} (\nabla^2 v_z)^* + \frac{1}{Fr} g^*$$

Non-dimensional Continuity Equation

$$\frac{\partial v_x^*}{\partial x^*} + \frac{\partial v_y^*}{\partial y^*} + \frac{\partial v_z^*}{\partial z^*} = 0$$

$\frac{Dv_z}{Dt} = \frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z}$

$Pe = PrRe = \frac{\hat{C}_p \mu \rho V D}{k \mu}$

$Pr = \frac{\hat{C}_p \mu}{k}$

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Review of dimensional analysis

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The D.A. goes with a particular problem (a particular physics)

steady heat transfer

**Forced Convection Heat Transfer**



Linear driving force model  $\left| \frac{q_x}{A} \right| = h|T_1 - T_0|$

Apply at the interface:

$$(2\pi RL)(h)(T_1 - T_0) = Q = \iint_S [\hat{e}_r \cdot \vec{q}]_{surface} dS$$

$$(2\pi RL)(h)(T_1 - T_0) = Q = \int_0^{2\pi} \int_0^L -k \frac{\partial T}{\partial r} \Big|_{r=R} R dz d\theta$$

Now, non-dimensionalize this expression as well.

Here, the "engineering property of interest" is the heat transferred across the boundary,  $Q$ .

Yields correlations for nondimensional heat transfer coefficient,  $h$

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The heat transferred from the fluid (LHS) equals the heat transferred into the wall (RHS).

steady heat transfer

**LHS**

$$h(\cancel{\pi DL})(\cancel{T_1 - T_0}) = \int_0^{2\pi} \int_0^{L/D} -k \frac{\partial T^*}{\partial r^*} \Big|_{r^*=1/2} \frac{(\cancel{T_1 - T_0}) \cancel{D^2}}{2} dz^* d\theta$$

**RHS**

$$2\pi \left( \frac{hD}{k} \right) \left( \frac{L}{D} \right) = \int_0^{2\pi} \int_0^{L/D} \frac{\partial T^*}{\partial r^*} \Big|_{r^*=1/2} dz^* d\theta$$

This is a function of Re and Pr through fluid  $\nu$  distribution and energy balance

**Nusselt number, Nu**  
(dimensionless heat-transfer coefficient)

$$Nu = Nu \left( T^*, \frac{L}{D} \right)$$

one additional dimensionless group

The engineering quantity of interest produces Nu

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Dimensional Analysis in Mass Transfer Review of dimensional analysis

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The D.A. produces:  
 $Nu = Nu \left( Re, Pr, \frac{L}{D} \right)$

steady heat transfer

Complex Heat Transfer – Dimensional Analysis CM3110 REVIEW

According to our **dimensional analysis** calculations, the dimensionless heat transfer coefficient should be found to be a function of ~~four~~ <sup>three</sup> dimensionless groups:

no free surfaces

Peclet number

$$Pe \equiv \frac{\rho \hat{c}_p V D}{k} = \frac{\hat{c}_p \mu \rho V D}{k \mu}$$

Prandtl number

$$Pr \equiv \frac{\hat{c}_p \mu}{k}$$

$$Nu = Nu \left( Re, Pr, Fr, \frac{L}{D} \right)$$

Now, do the experiments.

Can only know if the D.A. is right, if the D.A. works.

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Review of dimensional analysis

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The experiments produce (turbulent flow):

$$Nu = 0.027Re^{0.8}Pr^{\frac{1}{3}}\left(\frac{\mu_b}{\mu_w}\right)^{0.14}$$

steady heat transfer

Complex Heat Transfer – Dimensional Analysis


Now, do the experiments.

Forced Convection Heat Transfer

- Build apparatus (*several* actually, with different D, L)
- Run fluid through the inside (at different  $v$ ; for different fluids  $\rho, \mu, \hat{C}_p, k$ )
- Measure  $T_{bulk}$  on inside;  $T_{wall}$  on inside
- Measure rate of heat transfer,  $Q$
- Calculate  $h$ :  $|Q| = hA|T_{bulk} - T_{wall}|$
- Report  $h$  values in terms of dimensionless correlation:

$$Nu = \frac{hD}{k} = f\left(Re, Pr, \frac{L}{D}\right)$$

It should only be a function of these dimensionless numbers (if our Dimensional Analysis is correct....)

AND IT WORKS!

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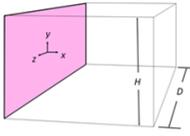
Dimensional Analysis in Mass Transfer
Review of dimensional analysis

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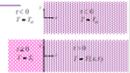
We also applied D.A. to **unsteady heat transfer**:

Unsteady heat transfer

Let's nondimensionalize the governing equations and BCs.  
Let's sort out the various cases.



1D Heat Transfer: Unsteady State

Unsteady State Heat Conduction in a Semi-Infinite Slab


$$\frac{\partial T}{\partial t} = \frac{k}{\rho \hat{C}_p} \left( \frac{\partial^2 T}{\partial x^2} \right) = \alpha \left( \frac{\partial^2 T}{\partial x^2} \right)$$

Initial condition:  $t = 0 \quad T = T_0 \quad \forall x$

Boundary conditions:

$x = 0 \quad \frac{q_x}{A} = -k \frac{dT}{dx} = h(T_1 - T) \quad t > 0$

$x = \infty \quad T = T_0 \quad \forall t$

thermal diffusivity  $\alpha = \frac{k}{\rho \hat{C}_p}$

(Review:  
**How did we do this before?**)

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Dimensional Analysis in Mass Transfer Review of dimensional analysis

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We also applied D.A. to unsteady heat transfer:

Unsteady heat transfer

We'll modify our solution for  
**Convective Heat Transfer**



Pipe flow

Dimensional Analysis

non-dimensional variables:

time: $t^* \equiv \frac{tV}{D}$	position: $r^* \equiv \frac{r}{D}$ $z^* \equiv \frac{z}{D}$	velocity: $v_r^* \equiv \frac{v_r}{V}$ $v_z^* \equiv \frac{v_z}{V}$	driving force: $P^* \equiv \frac{P}{\rho V^2}$ $g_z^* \equiv \frac{g_z}{g}$
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Energy

non-dimensional variables:

position: $r^* \equiv \frac{r}{D}$ $z^* \equiv \frac{z}{D}$	temperature: $T^* \equiv \frac{T - T_0}{(T_1 - T_0)}$	source: $S^* \equiv \frac{S}{S_0}$
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Slight problem: We need to nondimensionalize  $t$  for the unsteady case also, but there is **no characteristic velocity** in thermal conduction in a solid.

Had to adjust the characteristic time

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Dimensional Analysis in Mass Transfer Review of dimensional analysis

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We also applied D.A. to unsteady heat transfer:

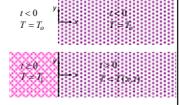
Unsteady heat transfer

Dimensional Analysis, Unsteady State Convection

Non-dimensionalize (eqns, BCs)

$$\frac{\partial T}{\partial t} = \alpha \left( \frac{\partial^2 T}{\partial x^2} \right)$$

$$q_x = -k \frac{\partial T}{\partial x} = hA(T_1 - T)$$



non-dimensional variables:

position:

$$x^* \equiv \frac{x}{D}$$

temperature:

$$Y \equiv \frac{(T_1 - T)}{(T_1 - T_0)}$$

time:

$$t^* \equiv \frac{\alpha t}{D^2}$$

This dimensionless time is called Fourier number Fo.

Fo – Fourier Number =  $\frac{\alpha t}{D^2}$

Had to adjust the characteristic time

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Dimensional Analysis in Mass Transfer
Review of dimensional analysis

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We also applied D.A. to unsteady heat transfer:

Unsteady heat transfer

In dimensionless form, we see that this problem reduces to

$$Y = Y\left(\frac{x}{D}, Fo, Bi\right)$$

Dimensionless quantities:

$$Y = \frac{(T_1 - T)}{(T_1 - T_0)}$$

$$t^* = Fo = \frac{\alpha t}{D^2}$$

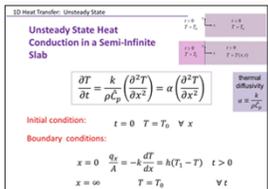
$$x^* = \frac{x}{D}$$

$$Bi = \frac{hD}{k}$$

**Y** (dimensionless temperature interval)

**Fourier number** (dimensionless time)

**Biot number** (pronounced BEE-OH)  
Ratio of heat transfer resistance at the boundary to resistance in the solid. This is a transport issue.



(Heissler charts)

AND IT WORKS!

END REVIEW

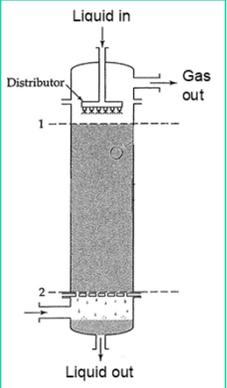
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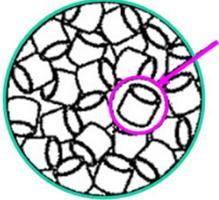
Dimensional Analysis in Mass Transfer

Returning to our question:

## What do we do to understand complex mass transfer?

1. Find a simple problem that allows us to identify the physics
2. Non-dimensionalize:
  - a. Choose characteristic values
  - b. Produce a non-dimensional governing equation
  - c. Produce a non-dimensional engineering quantity of interest
3. Explore that problem
4. Take data and correlate (confirm D.A. for chosen problem)
5. Solve real problems with the correlation

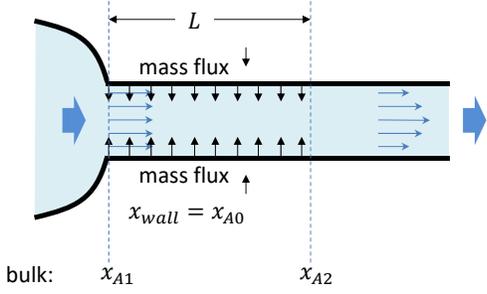




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Dimensional Analysis in Mass Transfer

**Example 12:** What is the mass transfer through the walls of a permeable tube (laminar or turbulent flow)?



Assumptions:

1. Isothermal
2. Steady flow
3. Uniform inlet composition  $x_{A1}$
4. Constant interfacial liquid composition of  $x_{A0}$
5.  $\rho, \mu, c, D_{AB}$  all constant
6. Radial mass flux (negative)

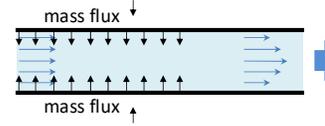
$$\begin{aligned} \text{Total mass in} &= \int_0^L \int_0^{2\pi} +cD_{AB} \left. \frac{\partial x_A}{\partial r} \right|_{r=R} R d\theta dz \\ &= k_x (2\pi RL)(x_{A0} - x_{A1}) \end{aligned}$$

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Dimensional Analysis in Mass Transfer

**Forced Convection Mass Transfer**

**Pipe flow**



$$k_x (2\pi RL)(x_{A0} - x_{A1}) = \int_0^L \int_0^{2\pi} +cD_{AB} \left. \frac{\partial x_A}{\partial r} \right|_{r=R} R d\theta dz$$

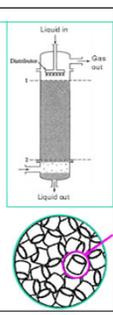
Next?

Dimensional Analysis in Mass Transfer

Returning to our question:

What do we do to understand complex mass transfer?

1. Find a simple problem that allows us to identify the physics
2. Non-dimensionalize:
  - a. Choose characteristic values
  - b. Produce a non-dimensional governing equation
  - c. Produce a non-dimensional engineering quantity of interest
3. Explore that problem
4. Take data and correlate (confirm D.A. for chosen problem)
5. Solve real problems with the correlation



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Dimensional Analysis in Mass Transfer

### Forced Convection Mass Transfer

**Pipe flow**

non-dimensional variables:

time:

$$t^* \equiv \frac{tV}{D}$$

position:

$$r^* \equiv \frac{r}{D}$$

$$z^* \equiv \frac{z}{D}$$

velocity:

$$v_z^* \equiv \frac{v_z}{V}$$

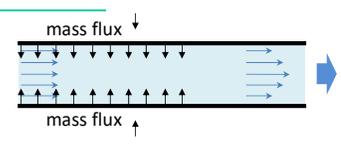
$$v_r^* \equiv \frac{v_r}{V}$$

$$v_\theta^* \equiv \frac{v_\theta}{V}$$

driving force:

$$P^* \equiv \frac{P}{\rho V^2}$$

$$g_z^* \equiv \frac{g_z}{g}$$



- Choose “typical” values (*scale factors*)
- Use them to scale the equations
- Deduce which terms dominate

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Dimensional Analysis in Mass Transfer

### Forced Convection Mass Transfer

**Species A Mass**

Microscopic species A mass balance (no reaction):

$$c \left( \frac{\partial x_A}{\partial t} + v_r \frac{\partial x_A}{\partial r} + \frac{v_\theta}{r} \frac{\partial x_A}{\partial \theta} + v_z \frac{\partial x_A}{\partial z} \right) = cD_{AB} \left( \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial x_A}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 x_A}{\partial \theta^2} + \frac{\partial^2 x_A}{\partial z^2} \right)$$

non-dimensional variables:

position:

$$r^* \equiv \frac{r}{D}$$

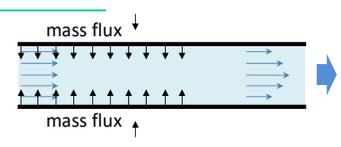
$$z^* \equiv \frac{z}{D}$$

composition

$$x_A^* = \frac{(x_A - x_{A0})}{(x_{A1} - x_{A0})}$$

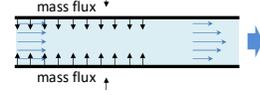
Choose:

$x_A$  – use a characteristic interval



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Dimensional Analysis in Mass Transfer—Forced Convection



**Non-dimensional Species A Mass Equation**

$$\left( \frac{\partial x_A^*}{\partial t^*} + v_r^* \frac{\partial x_A^*}{\partial r^*} + \frac{v_\theta^*}{r^*} \frac{\partial x_A^*}{\partial \theta} + v_z^* \frac{\partial x_A^*}{\partial z^*} \right) = \frac{1}{\text{Pe}_m} \left( \frac{1}{r^*} \frac{\partial}{\partial r^*} \left( r^* \frac{\partial x_A^*}{\partial r^*} \right) + \frac{1}{r^{*2}} \frac{\partial^2 x_A^*}{\partial \theta^2} + \frac{\partial^2 x_A^*}{\partial z^{*2}} \right)$$

**Non-dimensional Navier-Stokes Equation**

$$\frac{Dv_z^*}{Dt^*} = -\frac{\partial P^*}{\partial z^*} + \frac{1}{\text{Re}} (\nabla^2 v_z^*) + \frac{1}{\text{Fr}} g^*$$

**Non-dimensional Continuity Equation**

$$\frac{\partial v_x^*}{\partial x^*} + \frac{\partial v_y^*}{\partial y^*} + \frac{\partial v_z^*}{\partial z^*} = 0$$

$$\text{Pe}_m = \text{ReSc} = \frac{VD}{D_{AB}}$$

$$\text{Sc} = \frac{\mu}{\rho D_{AB}}$$

Schmidt number  
(a material property)

$$\frac{Dv_z}{Dt} \equiv \frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z}$$

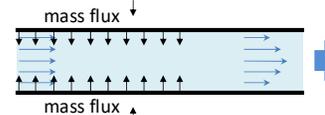
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Dimensional Analysis in Mass Transfer

**Forced Convection Mass Transfer**

**Pipe flow** Now, non-dimensionalize this expression as well.



$$k_x(2\pi RL)(x_{A0} - x_{A1}) = \int_0^L \int_0^{2\pi} + cD_{AB} \frac{\partial x_A}{\partial r} \Big|_{r=R} R d\theta dz$$

$$\text{Sh} = \text{Nu}_{AB} = \frac{k_x D}{cD_{AB}} = \frac{1}{2\pi \left(\frac{L}{D}\right)} \int_0^L \int_0^{2\pi} - \frac{\partial x_A^*}{\partial r} \Big|_{r^*=\frac{1}{2}} d\theta dz^*$$

This is a function of Re and Sc through fluid  $v$  distribution and species A mass balance

**Sherwood number, Sh**  
(dimensionless mass-transfer coefficient)

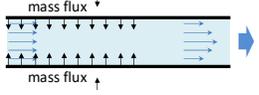
$$\text{Sh} = \text{Sh} \left( x_A^*, \frac{L}{D} \right)$$

one additional dimensionless group

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Dimensional Analysis in Mass Transfer



According to our **dimensional analysis** calculations, the dimensionless heat transfer coefficient should be found to be a function of three dimensionless groups:

Peclet number

$$Pe_m = ReSc = \frac{VD}{D_{AB}}$$

Schmidt number

$$Sc = \frac{\mu}{\rho D_{AB}}$$

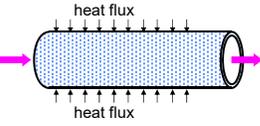
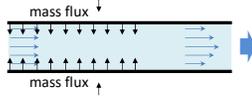
$$Sh = Sh \left( Re, Sc, \frac{L}{D} \right)$$

Now, do the experiments.

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Dimensional Analysis in Mass Transfer

**Note** this development has been exactly the same as a related heat transfer development:

Complex Heat Transfer – Dimensional Analysis

CM3110  
REVIEW

According to our **dimensional analysis** calculations, the dimensionless heat transfer coefficient should be found to be a function of four dimensionless groups:

↙ three

no free surfaces

Peclet number

$$Pe = \frac{\rho c_p V D}{k} = \frac{c_p \mu \rho V D}{k \mu}$$

Prandtl number

$$Pr = \frac{c_p \mu}{k}$$

$$Nu = Nu \left( Re, Pr, \frac{L}{D} \right)$$

Now, do the experiments.

Dimensional Analysis in Mass Transfer

According to our **dimensional analysis** calculations, the dimensionless heat transfer coefficient should be found to be a function of three dimensionless groups:

Peclet number

$$Pe_m = ReSc = \frac{VD}{D_{AB}}$$

Schmidt number

$$Sc = \frac{\mu}{\rho D_{AB}}$$

$$Sh = Sh \left( Re, Sc, \frac{L}{D} \right)$$

Now, do the experiments.

In many cases, heat and mass transfer are **analogous**

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## Dimensional Analysis

**Dimensionless numbers from the Equations of Change** (microscopic balances)

**momentum** Non-dimensional Navier-Stokes Equation

$$\left(\frac{\partial v_z^*}{\partial t^*} + \underline{v}^* \cdot \nabla^* v_z^*\right) = -\frac{\partial P^*}{\partial z^*} + \frac{1}{\text{Re}}(\nabla^{*2} v_z^*) + \frac{1}{\text{Fr}}g^*$$

**energy** Non-dimensional Energy Equation

$$\left(\frac{\partial T^*}{\partial t^*} + \underline{v}^* \cdot \nabla^* T^*\right) = \frac{1}{\text{RePr}}(\nabla^{*2} T^*) + S^*$$

**mass** Non-dimensional Continuity Equation (species A)

$$\left(\frac{\partial x_A^*}{\partial t^*} + \underline{v}^* \cdot \nabla^* x_A^*\right) = \frac{1}{\text{ReSc}}(\nabla^{*2} x_A^*)$$

These numbers tell us about the relative importance of the terms they precede.

**Re** – Reynolds  
**Fr** – Froude

**Pe** – Péclet<sub>n</sub> = RePr  
**Pr** – Prandtl

**Pe** – Péclet<sub>m</sub> = ReSc  
**Sc** – Schmidt

ref: BSL1, p581, 644

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## Dimensional Analysis

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**mass** Non-dimensional Continuity Equation (species A)

$$\left(\frac{\partial x_A^*}{\partial t^*} + \underline{v}^* \cdot \nabla^* x_A^*\right) = \frac{1}{\text{ReSc}}(\nabla^{*2} x_A^*)$$

These numbers tell us about the relative importance of the terms they precede in the governing equations.

**Re** – Reynolds  
**Fr** – Froude

**Pe** – Péclet<sub>n</sub> = RePr  
**Pr** – Prandtl

**Pe** – Péclet<sub>m</sub> = ReSc  
**Sc** – Schmidt

ref: BSL1, p581, 644

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Oops! This is dimensionless  $\underline{v}$ , NOT molar average velocity, sorry!

## Dimensionless Numbers

Dimensionless numbers from the **Equations of Change**

**Re** – Reynolds =  $\frac{\rho VD}{\mu} = \frac{VD}{\nu}$

**Fr** – Froude =  $\frac{v^2}{gD}$

**Pe** – Péclet<sub>h</sub> = **RePr** =  $\frac{\hat{c}_p \rho VD}{k} = \frac{VD}{\alpha}$

**Pe** – Péclet<sub>m</sub> = **ReSc** =  $\frac{VD}{D_{AB}}$

**Pr** – Prandtl =  $\frac{\hat{c}_p \mu}{k} = \frac{\nu}{\alpha}$

**Sc** – Schmidt = **LePr** =  $\frac{\mu}{\rho D_{AB}} = \frac{\nu}{D_{AB}}$

**Le** – Lewis =  $\frac{\alpha}{D_{AB}}$

These numbers tell us about the **relative importance of the terms** they precede in the microscopic balances (*scenario properties*).

These numbers compare the **magnitudes of the diffusive transport coefficients**  $\nu, \alpha, D_{AB}$  (*material properties*).

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## Dimensionless Numbers

Dimensionless numbers from the **Equations of Change**

**Re** – Reynolds =  $\frac{\rho VD}{\mu} = \frac{VD}{\nu}$

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These numbers tell us about the **relative importance of the terms** they precede in the microscopic balances (*scenario properties*).

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**Transport coefficients**

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## Dimensional Analysis

**Dimensionless numbers from the Engineering Quantities of Interest**

**momentum** × Dimensionless Force on the Wall (Drag)

$$f = \frac{1}{\pi L k_e} \int_0^{\frac{D}{2}} \int_0^{2\pi} \left( \frac{\partial v_z}{\partial r^*} \right) \Big|_{r^*=\frac{1}{2}} d\theta dz^*$$

**energy** × Newton's Law of Cooling

$$Nu = \frac{1}{2\pi L / D} \int_0^{\frac{D}{2}} \int_0^{2\pi} \left( \frac{\partial T^*}{\partial r^*} \right) \Big|_{r^*=\frac{1}{2}} dz^* d\theta$$

**mass xfer** × Dimensionless Mass Transfer Coefficient

$$Sh = \frac{1}{2\pi L k_c} \int_0^{\frac{D}{2}} \int_0^{2\pi} \left( -\frac{\partial x_A^*}{\partial r^*} \right) \Big|_{r^*=\frac{1}{2}} d\theta dz^*$$

These numbers are defined to help us build **transport data correlations** based on the fewest number of grouped (dimensionless) variables (*scenario property*).

$f$  – Friction Factor (Fanning)

$\frac{L}{D}$  – Aspect Ratio

$$f = \frac{\mathcal{F}_{drag}}{\left(\frac{1}{2}\rho V^2\right) A_c}$$

$Nu$  – Nusselt

$\frac{L}{D}$  – Aspect Ratio

$$Nu = \frac{hD}{k}$$

$St_h = \frac{h}{\rho V \hat{c}_p} = \frac{Nu}{RePr}$

$Sh$  – Sherwood

$\frac{L}{D}$  – Aspect Ratio

$$Sh = \frac{k_c D}{D_{AB}}$$

$St_m = \frac{k_c}{V} = \frac{Sh}{ReSc}$

$St$  – Stanton

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momentum  
energy  
mass

## Dimensionless Numbers

$Re$  – Reynolds =  $\frac{\rho VD}{\mu} = \frac{VD}{\nu}$

$Fr$  – Froude =  $\frac{V^2}{gD}$

$Pe$  – Péclet<sub>h</sub> =  $RePr = \frac{\hat{c}_p \rho VD}{k} = \frac{VD}{\alpha}$

$Pe$  – Péclet<sub>m</sub> =  $ReSc = \frac{VD}{D_{AB}}$

These numbers from the governing equations tell us about the relative importance of the terms they precede in the microscopic balances (*scenario properties*).

$Pr$  – Prandtl =  $\frac{\hat{c}_p \mu}{k} = \frac{\nu}{\alpha}$

$Sc$  – Schmidt =  $LePr = \frac{\mu}{\rho D_{AB}} = \frac{\nu}{D_{AB}}$

$Le$  – Lewis =  $\frac{\alpha}{D_{AB}}$

These numbers compare the magnitudes of the diffusive transport coefficients  $\nu, \alpha, D_{AB}$  (*material properties*).

$f$  – Friction Factor =  $\frac{\mathcal{F}_{drag}}{\left(\frac{1}{2}\rho V^2\right) A_c}$

$Nu$  – Nusselt =  $\frac{hD}{k}$

$Sh$  – Sherwood =  $\frac{k_c D}{D_{AB}}$

These numbers are defined to help us build transport data correlations based on the fewest number of grouped (dimensionless) variables (*scenario properties*).

$St_h = Nu/Pe_h, St_m = Sh/Pe_m$  – Stanton

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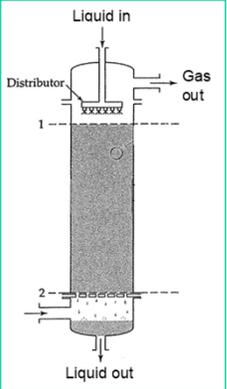
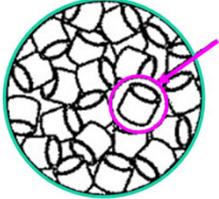
Dimensional Analysis in Mass Transfer

**Steps to produce correlations**

*Returning to our question:*

### What do we do to understand complex mass transfer?

1. Find a simple problem that allows us to identify the physics
2. Non-dimensionalize:
  - a. Choose characteristic values
  - b. Produce a non-dimensional governing equation
  - c. Produce a non-dimensional engineering quantity of interest
3. Explore that problem
4. Take data and correlate (confirm D.A. for chosen problem)
5. Solve real problems with the correlation

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Dimensional Analysis in Mass Transfer

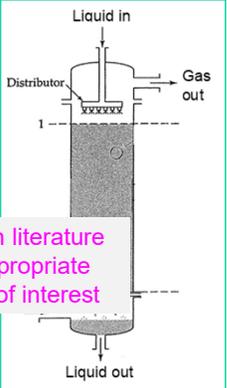
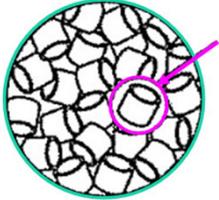
**Steps to use correlations**

*Returning to our question:*

### What do we do to understand complex mass transfer?

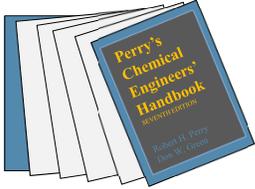
1. Find a simple problem that allows us to identify the physics
- ~~2. Non-dimensionalize.~~
  - ~~a. Choose characteristic values~~
  - ~~b. Produce a non-dimensional governing equation~~
  - ~~c. Produce a non-dimensional engineering quantity of interest~~
- ~~3. Explore that problem~~
- ~~4. Take data and correlate (confirm D.A. for chosen problem)~~
- ~~5. Solve real problems with the correlation (be sure to validate choice)~~

**2. Determine which literature correlation is appropriate for the problem of interest**

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Dimensional Analysis in Mass Transfer



## Perry's Chemical Engineers' Handbook

7<sup>th</sup> edition (1997)  
Robert H. Perry  
Don W. Green

*See also:*  
(Green and Southard, 9<sup>th</sup> edition, 2019)

**Section 5: Heat and Mass Transfer**

**Authors of Mass Transfer:**  
Phillip C. Wankat  
Kent S. Knaebel

**Table 5-21: Correlations for Mass Transfer:** (pp 5-59 thru 5-77)

- From fluid to plate
- To a falling film
- In pipes and ducts
- Past submerged objects
- To/from bubbles, drops
- In agitated systems
- In fixed and fluidized beds
- In packed two-phase contactors (absorption, distillation, cooling towers)

(T)-theoretical  
(S)-semi-empirical  
(E)-empirical

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Dimensional Analysis in Mass Transfer

## Advice from Wankat and Knaebel

$$Sh = Sh(Re, Sc)$$

1. Because of its importance, there are many studies of mass transfer in the literature
2. For simple geometries, theoretical results are obtainable (T)
3. For very complex systems, only empirical (E) forms can be found
4. Theoretical correlations can be "improved" by fitting to data, resulting in a semi-empirical correlation (S)
5. The major limits and constraints are listed in Perry's Table 5-21; many details are not included, however
6. Readers are *strongly encouraged* to check the references before using the correlations; look for comparisons to actual data
7. Even authoritative sources have typos

(Perry's, 7<sup>th</sup> ed, p 5-58)



**Perry's Chemical Engineers' Handbook**

7<sup>th</sup> edition (1997)  
Robert H. Perry  
Don W. Green

*See also:*  
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Dimensional Analysis in Mass Transfer

**MORE Advice from Wankat and Knaebel**

**$Sh = Sh(Re, Sc)$**

When there are several correlations that are applicable (which often happens), how do we choose?

1. Determine which correlations are closest to the situation under study (similarity of geometries, checking the range of dimensionless numbers and other parameters)
2. Check to see if correlations under consideration have been compared in the literature, both to each other, and to data
3. Check for “rules of thumb” shared by experts

(Perry's, 7<sup>th</sup> ed, p 5-58 through 5-60)



**Perry's Chemical Engineers' Handbook**  
7<sup>th</sup> edition (1997)  
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Dimensional Analysis in Mass Transfer

**MORE Advice from Wankat and Knaebel**

**$Sh = Sh(Re, Sc)$**

**Rules of Thumb**

1. If arithmetic concentration difference was used to determine  $k$  for the correlation, that should only be used in such an expression
2. Semi-empirical correlations are often preferred to empirical (do *not* extrapolate empirical) or purely theoretical (can be far off; assumptions)
3. Correlations with a broader data base are preferred
4. Heat/mass transfer analogy is pretty good within its bounds; good heat transfer data (without radiation) can often be used to predict mass-transfer coefficients
5. Recent data is preferred over older data
6. With complex geometries,  $k_y a$  (or HTU) correlations are more accurate than  $k_y$  correlations
7. **If a mass-transfer correlation looks too good to be true, it probably is.**

(Perry's, 7<sup>th</sup> ed, p 5-60)



**Perry's Chemical Engineers' Handbook**  
7<sup>th</sup> edition (1997)  
Robert H. Perry  
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Section 5: Heat and Mass Transfer  
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### Routes to Mass Transfer Correlations

#### Theoretical

- Based on a model of the situation; can be solved for flux, and thus for  $k_x$
- All models have assumptions; how good the assumptions are determine how good the correlations are
- The heat-transfer analogy counts as a theoretical pathway

#### Semi-empirical

- Shortcomings in theoretical models may be “fixed” by adjusting a theoretically derived coefficient or exponent to achieve a better fit to the data
- The theoretical underpinnings give some hope that extrapolation outside the range of the data fit may be valid

#### Empirical

- A pure fit to the data is a convenient way to use data directly in downstream calculations
- With no theoretical underpinnings it is extremely unwise to use empirical correlations outside the range of the data fit

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### Routes to Mass Transfer Correlations

#### Theoretical

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The Heat/Mass Transfer Analogy

**Routes to Mass Transfer Correlations**

**Theoretical**

- Based on a model of the situation; can be solved for flux, and thus for  $k_x$
- All models have assumptions; how good the assumptions are determine how good the correlations are
- The heat-transfer analogy counts as a theoretical pathway

**Theoretical Pathway to Mass Transfer Coefficients:**

*The Heat/Mass Transfer Analogy*

**mass**

**Example:** A spherical medical pill dissolves slowly in a stagnant fluid (water). What is the mass transfer coefficient  $k_c$ ? What is the Sherwood number for this situation?

**heat**

**Example:** A spherical pellet of reacting solid slowly emits heats steadily into a stagnant fluid such that the surface temperature is constant. What is the heat transfer coefficient  $h$ ? What is the Nusselt number for this situation?

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The Heat/Mass Transfer Analogy

**Routes to Mass Transfer Correlations**

**Theoretical**

- Based on a model of the situation; can be solved for flux, and thus for  $k_x$
- All models have assumptions; how good the assumptions are determine how good the correlations are
- The heat-transfer analogy counts as a theoretical pathway

**heat**

**Example:** A spherical pellet of reacting solid slowly emits heats steadily into a stagnant fluid such that the surface temperature is constant. What is the heat transfer coefficient  $h$ ? What is the Nusselt number for this situation?

$T_\infty$        $T_R$        $R$       stagnant fluid

Solve.

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The Heat/Mass Transfer Analogy

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**mass**

**Example:** A spherical medical pill dissolves slowly in a stagnant fluid (water). What is the mass transfer coefficient  $k_c$ ? What is the Sherwood number for this situation?

$x_{A\infty}$

$x_{AR}$

$R$

stagnant fluid

Solve.

BSI2 p321, problem 10B.1 53 © Faith A. Morrison, Michigan Tech U.

The Heat/Mass Transfer Analogy

Routes to Mass Transfer Correlations

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**Theoretical Pathway to Mass Transfer Coefficients:**

*The Heat/Mass Transfer Analogy*

**Results for transfer from sphere to stagnant fluid:**

- Sh = Nu = 2
- Limited to low mass transfer rates ( $v^* \approx 0$ )
- At low mass transfer rates and stagnant fluid,  $J_A^* = \underline{N}_A$  and  $j_A = \underline{n}_A$ ; this makes it easy to convert units moles to mass

**Assumptions of the analogy between heat and mass**

- Constant physical properties
- Small net mass transfer rates
- No chemical reactions
- No viscous dissipation heating
- No absorption or emission of radiant energy
- No pressure diffusion, thermal diffusion, or forced diffusion

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The Heat/Mass Transfer Analogy

Dimensional Analysis in Mass Transfer

**Theoretical Pathway to Mass Transfer Coefficients:**

*The Heat/Mass Transfer Analogy*

**Results for transfer from sphere to stagnant fluid:**

- $Sh = Nu = 2$
- Limited to low mass transfer rates ( $v^* \approx 0$ )
- At low mass transfer rates and stagnant fluid,  $j_A^* = \dot{N}_A$  and  $j_A = \dot{n}_A$ ; this makes it easy to convert units moles to mass

**Routes to Mass Transfer Correlations**

**Theoretical**

- Based on a model of the situation; can be solved for flux, and thus for  $k_x$
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**Assumptions of the analogy between heat and mass**

1. Constant physical properties
2. Small net mass transfer rates
3. No chemical reactions
4. No viscous dissipation heating
5. No absorption or emission of radiant energy
6. No pressure diffusion, thermal diffusion, or forced diffusion

**Comment from the experts:**

“It would be very misleading to leave the impression that all mass transfer coefficients can be obtained from the analogous heat transfer coefficient correlations. For mass transfer we encounter a much wider variety of boundary conditions and other ranges of relevant variables.”

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Dimensional Analysis in Mass Transfer

**Routes to Mass Transfer Correlations**

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- Based on a model of the situation; can be solved for flux, and thus for  $k_x$
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**Semi-empirical**

- Shortcomings in theoretical models may be “fixed” by adjusting a theoretically derived coefficient or exponent to achieve a better fit to the data
- The theoretical underpinnings give some hope that extrapolation outside the range of the data fit may be valid

**Empirical**

- A pure fit to the data is a convenient way to use data directly in downstream calculations
- With no theoretical underpinnings it is extremely unwise to use empirical correlations outside the range of the data fit

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Dimensional Analysis in Mass Transfer

Routes to Mass Transfer Correlations

**Semi-Empirical**

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### Semi-Empirical Pathway to Mass Transfer Coefficients

Inspired by theoretical results and a model (a picture of how the mass transfer may be explained), correlations may be created that are then fine-tuned to match the data

For example, **Colburn’s extension of the Reynolds analogy**

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Dimensional Analysis in Mass Transfer

Routes to Mass Transfer Correlations

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### Reynolds Analogy, Colburn’s, Prandtl’s extensions

- Reynolds noted the similarities in mechanism between energy and momentum transfer
- He derived, for restrictive conditions ( $Pr = 1$ , no form drag), the following equation:
 
$$\frac{h}{\rho V_{\infty} \hat{C}_p} = St_h = \frac{f}{2} \quad \text{(Stanton number for heat transfer)}$$
- Coleburn modified the Reynolds result to work at more values of  $Pr$  and proposed the following:
 
$$St_h Pr^{2/3} = \frac{f}{2}$$
- This improved relationship does a better job of predicting heat transfer coefficients and
- Separating the turbulent core from the laminar sublayer in boundary layer flow allows it to be extended to mass transfer (Prandtl), resulting in a refined empirical correlation (WRF eqn 28-54)

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BSL2 p681; WRF p580

Dimensional Analysis in Mass Transfer

### Routes to Mass Transfer Correlations

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Dimensional Analysis in Mass Transfer

### Empirical Pathway to Mass Transfer Coefficients

Inspired by looking at data from a variety of systems, correlations may be created that are fine-tuned to match the data.

These may be based purely on dimensional analysis or there may be a model that the researchers have in mind.

Empirical models are judged by how accurately they represent the data.

**Routes to Mass Transfer Correlations**

**Empirical**

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Dimensional Analysis in Mass Transfer

### Chilton-Colburn Analogy

Inspired by semi-empirical analogies such as the Reynolds Analogy, define the “j factors”:

$$j_H \equiv \frac{Nu}{RePr^{1/3}} = \frac{h}{\rho \hat{C}_p V_\infty} \left( \frac{\hat{C}_p \mu}{k} \right)^{2/3}$$

$$j_M \equiv \frac{Sh}{ReSc^{1/3}} = \frac{k_x}{c V_\infty} \left( \frac{\mu}{\rho D_{AB}} \right)^{2/3}$$

Compare to data.

#### Routes to Mass Transfer Correlations

**Empirical**

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### Chilton-Colburn Analogy

$$j_H = j_M = \frac{f}{2}$$

*(f is the Fanning friction factor)*

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Dimensional Analysis in Mass Transfer

### Chilton-Colburn Analogy

$$j_H = j_M = \frac{f}{2}$$

*(f is the Fanning friction factor)*

#### Chilton-Colburn Analogy

$$j_H \equiv \frac{Nu}{RePr^{1/3}} = \frac{h}{\rho \hat{C}_p V_\infty} \left( \frac{\hat{C}_p \mu}{k} \right)^{2/3}$$

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#### Routes to Mass Transfer Correlations

**Empirical**

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- With no theoretical underpinnings it is extremely unwise to use empirical correlations outside the range of the data fit

**Conditions:**

- Exact for flat plates
- Satisfactory in other geometries as long as form drag is not present
- Relates convective heat and mass transfer
- Permits evaluation of one transfer coefficient through information obtained on another
- Experimentally validated for gases and liquids within the ranges  $0.60 \leq Sc \leq 2500, 0.6 \leq Pr \leq 100$
- Constant physical properties data

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### Dimensional Analysis in Mass Transfer

**Routes to Mass Transfer Correlations**

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## Final thoughts on Literature Mass-Transfer Coefficient Correlations

- Choose correlation carefully
- Check the original reference (how  $k_y$  defined, what are the assumptions, how well does it represent the data)
- With complex geometries,  $k_y a$  (or HTU) correlations are more accurate than  $k_y$  correlations

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### Mass Transport "Laws"

## We have 2 Mass Transport "laws"

Remaining Topics to round out our understanding of mass transport:

*Fick's law of diffusion*

}

1. Since we predict  $N_A$  with Fick's law, we can also predict a mass transfer coefficient  $k_y$  or  $k_c$ . *Relate  $k_c$  and  $D_{AB}$*
2. 1D Unsteady models can be solved (if good at math). *Solutions are analogous to heat transfer*

*Mass transfer coefficients*

}

3. Combine with macroscopic species A mass balance. *Model macroscopic processes, design units*
4. Are not material properties; rather, they are determined experimentally and specific to the situation (dimensional analysis and correlations).  $Sh = f(ReSc)$
5. Facilitate combining resistances into overall mass transfer coefficients,  $K_L, K_G$ , to be used in modeling unit operations

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Mass Transport "Laws"

**We now have 2 Mass Transport "laws"**

**Fick's Law of Diffusion**  $N_A = x_A(N_A + N_B) - cD_{AB} \nabla x_A$

Use: Combine with microscopic species A mass balance  
 Predicts flux  $N_A$  and composition distributions, e.g.  $x_A(x, y, z, t)$

1D Steady models can be solved

1D Unsteady models can be solved (if good at math)

2D steady and unsteady models can be solved by Comsol

Since we predict  $N_A$ , we can also predict a mass xfer coeff.  $k_y$  or  $k_c$

Diffusion coefficients are **material** properties (see tables)

Transport coefficient

---

**Linear-Driving-Force Model**  $|N_A| = k_y |x_{A,bulk} - x_A|$

Use: Combine with macroscopic species A mass balance  
 Predicts flux  $N_A$  but **not** composition distributions

May be used as a boundary condition in microscopic balances

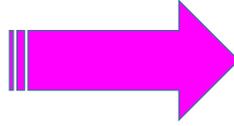
Mass-transfer-coefficients are **not material properties**

Rather, they are determined experimentally and specific to the situation (dimensional analysis and correlations)

Facilitate combining resistances into overall mass xfer coeffs.  $K_L, K_G$

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# Last topic



Overall Mass Transfer Coefficients

CM3110  
Transport II  
Part II: Diffusion and Mass Transfer

Michigan Tech



*Overall Mass Transfer Coefficients*

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